



Faculty of Civil Engineering
CALCULATION SHEET

Job No.	Sheet 1 of 10	Rev:
Job title		
Subject: Example 3 - Simply supported beam with lateral restraint at load application points		
Client: 3ed YEAR- Faculty of civil Engineering	Made by: SC1	Date:
	Checked by: Dr. G. Hallak	Date:

The beam shown in Figure 4.1 is laterally restrained at the ends and at the points of load application only. For the loading shown, design the beam in S275 steel.

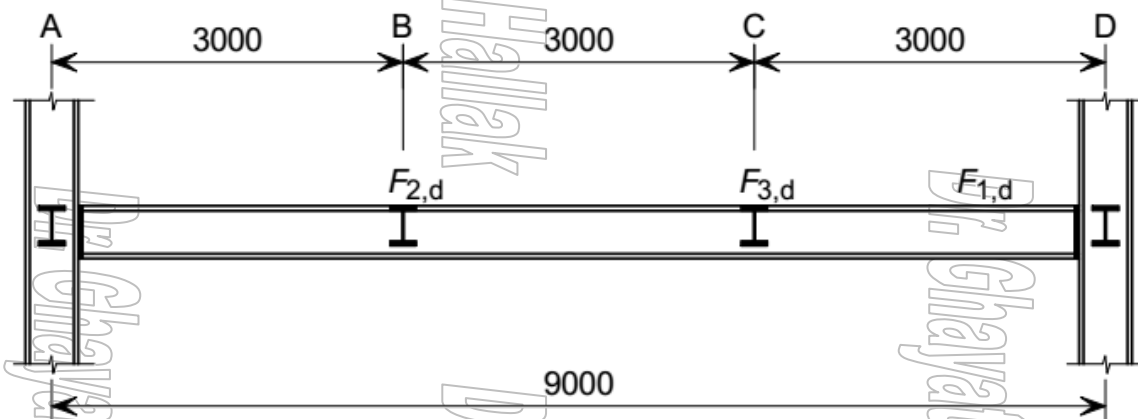


Figure 4.1

References are to BS EN 1993-1-1: 2005, including its National Annex, unless otherwise stated

1.2 Actions (loading)

1.2.1 Permanent actions

Uniformly distributed load (including self weight) $g = 3 \text{ kN/m}$

Concentrated load 1 $G1 = 40 \text{ kN}$

Concentrated load 2 $G2 = 20 \text{ kN}$

1.2.2 Variable actions

Concentrated load 1 $Q1 = 60 \text{ kN}$

Concentrated load 2 $Q2 = 30 \text{ kN}$

The variable actions are not due to storage and are not independent of each other.

1.2.3 Partial factors for actions

Partial factor for permanent actions $\gamma_G = 1.35$

Partial factor for variable actions $\gamma_Q = 1.50$

Reduction factor $\xi = 0.925$

1.2.4 Design values of combined actions for Ultimate Limit State

Use Expression (6.10) or the less favourable combination from Expression (6.10a) and (6.10b). The UK National Annex to BS EN 1990 allows the designer to choose which of those options to use.

BS EN 1990
A1.3.1(4)

Table
NA.A1.2(B)



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$$\gamma_{GJ, sup} G_{J, sup} + \gamma_{GJ, inf} G_{J, inf} + \gamma_{Q,1} \psi_{0,1} Q_1 + \gamma_{Q,1} \psi_{0,1} Q_i \quad (6.10a)$$

$$\xi \gamma_{GJ, sup} G_{J, sup} + \gamma_{GJ, inf} G_{J, inf} + \gamma_{Q,1} Q_1 + \gamma_{Q,1} \psi_{0,1} Q_i \quad (6.10b)$$

BS EN 1990
Table
NA.A1.2(B)

Use 6.10b:

UDL (including self weight)

$$F_{1,d} = \xi \gamma_G g = (0.925 \times 1.35 \times 3) = 3.7 \text{ kN/m}$$

Concentrated load 1

$$F_{2,d} = \xi \gamma_G G_1 + \gamma_Q Q_1 = (0.925 \times 1.35 \times 40) + (1.5 \times 60) = 140.0 \text{ kN}$$

Concentrated load 2

$$F_{3,d} = \xi \gamma_G G_2 + \gamma_Q Q_2 = (0.925 \times 1.35 \times 20) + (1.5 \times 30) = 70.0 \text{ kN}$$

1.3 Design bending moments and shear forces

The design bending moments and shear forces are shown in Figure 4.2

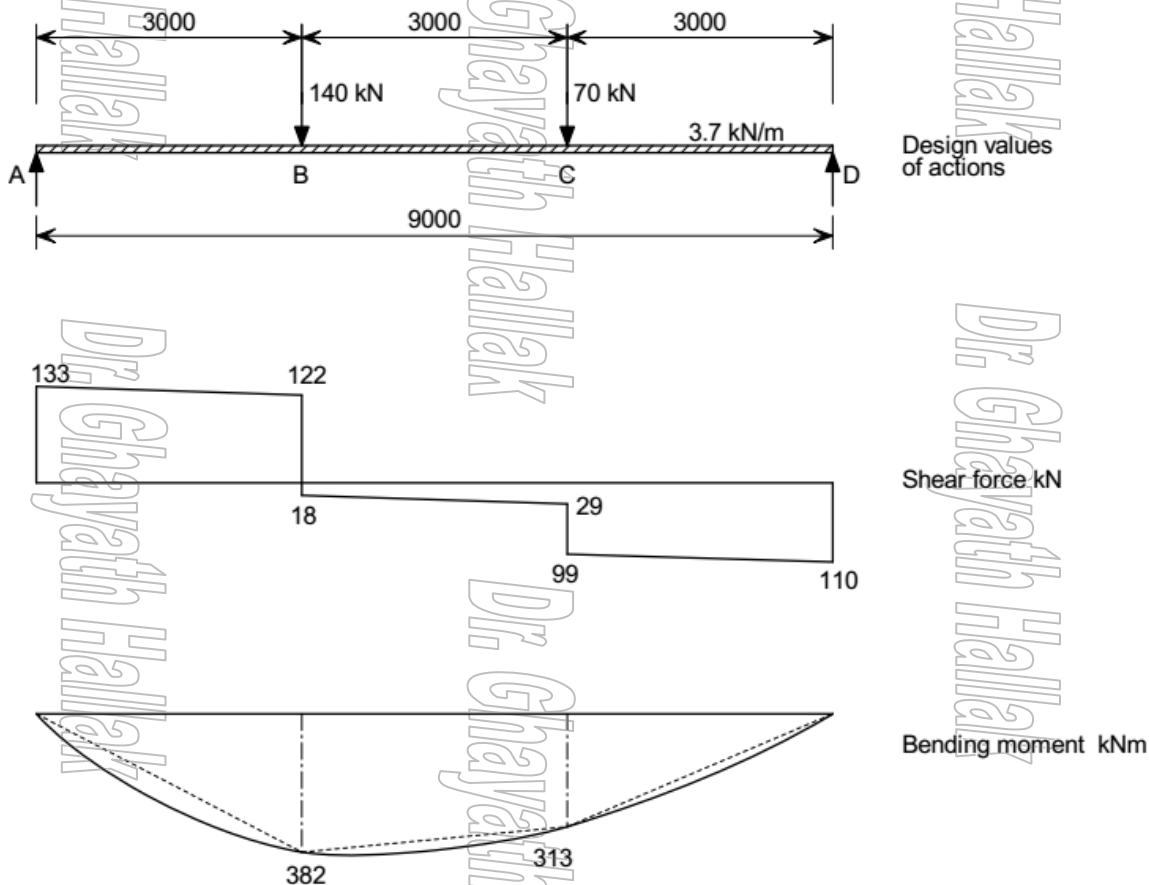


Figure 4.2



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1.4 Buckling length (L_{cr})

Since the beam is restrained at its ends and at the loading points, there are three segments to consider. From the bending moment diagram, it can be seen that the maximum bending moment occurs within segment B to C.

Therefore only this segment is considered.

BS EN 1993-1-1 does not give guidance for determining buckling lengths.

Therefore take the buckling length (L_{cr}) equal the span length between lateral restraints,

$$L_{cr} = 3000 \text{ mm}$$

1.5 Section properties

Trial section can be calculated as follows:

$$M_{b,R} = \Omega W_{pl,y} f_y / \gamma_{M0} = M_{Ed} \Rightarrow W_{pl,y} = M_{Ed} / (\Omega f_y / \gamma_{M0}) \Rightarrow$$

$$W_{pl,y} = 382 \times 10^6 / (0.80 \times 275) = 1736 \text{ cm}^3 \text{ where } \Omega = 0.65 \text{ to } 0.8$$

Chose from the UKB section tables a section has $W_{pl,y} > 1736 \text{ cm}^3$

Try section $457 \times 191 \times 82$ UKB in S275 with $W_{pl,y} = 1800 \text{ cm}^3$

From section property tables:

Depth	$h = 460.0 \text{ mm}$
Width	$b = 191.3 \text{ mm}$
Web thickness	$t_w = 9.9 \text{ mm}$
Flange thickness	$t_f = 16.0 \text{ mm}$
Root radius	$r = 10.2 \text{ mm}$
Depth between flange fillets	$d = 407.6 \text{ mm}$
Second moment of area, y -y axis	$I_y = 37\,100 \text{ cm}^4$
Second moment of area, z -z axis	$I_z = 1\,870 \text{ cm}^4$
Radius of gyration y-y axis	$i_y = 18.8 \text{ cm}$
Radius of gyration z-z axis	$i_z = 4.23 \text{ cm}$
Plastic modulus, y -y axis	$W_{pl,y} = 1\,830 \text{ cm}^3$
Plastic modulus, z -z axis	$W_{pl,z} = 304 \text{ cm}^3$
Area	$A = 104 \text{ cm}^2$
Buckling parameter	$U = 0.879$
Torsional constant	$I_T = 96.2 \text{ cm}^4$
Warping constant	$I_w = 0.922 \times 10^6 \text{ cm}^6$
Modulus of elasticity	$E = 210\,000 \text{ N/mm}^2$

Use $\Omega = 0.65$ if there are no lateral restraints between supports

And $\Omega = 0.8$ if lateral restraints between supports were provided



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For S275 steel and $t \leq 16$ mm
Yield strength

$$f_y = R_{eH} = 275 \text{ N/mm}^2$$

BS EN
10025-2
Table 7

1.6 Cross section classification

$$\varepsilon = \sqrt{\frac{235}{f_y}} = \sqrt{\frac{235}{275}} = 0.92$$

Outstand of compression flange

$$c = \frac{b - t_w - 2r}{2} = \frac{191.3 - 9.9 - (2 \times 10.2)}{2} = 80.50 \text{ mm}$$

$$\frac{c}{t_f} = \frac{80.5}{16.0} = 5.03$$

The limiting value for Class 1 is $\frac{c}{t_f} \leq 9\varepsilon = 9 \times 0.92 = 8.28$

$$5.03 < 8.28$$

Therefore the flange is Class 1 under compression.

Web subject to bending

$$c = d = 407.6 \text{ mm}$$

$$\frac{c}{t_w} = \frac{407.6}{9.9} = 41.17$$

The limiting value for Class 1 is $\frac{c}{t_w} \leq 72\varepsilon = 72 \times 0.92 = 66.24$

$$41.17 < 66.24$$

Therefore the web is Class 1 under bending.

Therefore the section is Class 1 under bending.

Table 5.2

1.7 Partial factors for resistance

$$\gamma_{M0} = 1.0$$

$$\gamma_{M1} = 1.0$$

NA.2.15



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1.8 Cross-sectional resistance

1.8.1 Shear buckling

The shear buckling resistance for webs should be verified according to Section 5 of BS EN 1993-1-5 if:

$$\frac{h_w}{t_w} > \frac{72\varepsilon}{\eta}$$

$$\eta = 1.0$$

$$h_w = h - 2t_f = 460.0 - (2 \times 16.0) = 428.0 \text{ mm}$$

$$\frac{h_w}{t_w} = \frac{428.0}{9.9} = 43.23$$

$$\frac{72\varepsilon}{\eta} = \frac{72 \times 0.92}{1.0} = 66.2$$

$$43.23 < 66.2$$

Therefore the shear buckling resistance of the web does not need to be verified.

1.8.2 Shear resistance

Verify that:

$$\frac{V_{Ed}}{V_{c,Rd}} \leq 1.0$$

$$V_{c,Rd} = V_{pl,Rd} = \frac{A_v \times f_y}{\gamma_{M0} \times \sqrt{3}}$$

$$A_v = A - 2t_f b + (t_w + 2r) t_f \geq \eta h_w t_w$$

$$A_v = 104 \times 10^2 - (2 \times 191.3 \times 16.0) + 16.0 \times [9.9 + (2 \times 10.2)] = 4763.2 \text{ mm}^2$$

$$\eta h_w t_w = 1.0 \times 428 \times 9.9 = 4237.2 \text{ mm}^2$$

Therefore,

$$A_v = 4763.2 \text{ mm}^2$$

$$V_{c,Rd} = V_{pl,Rd} = \frac{4763.2 \times 275}{1.0 \times \sqrt{3}} = 756 \text{ kN}$$

Maximum design shear occurs at A, therefore the design shear is

$$V_{A,Ed} = 133 \text{ kN}$$

$$\frac{V_{Ed}}{V_{c,Rd}} = \frac{133}{756} = 0.18 \leq 1.0$$

6.2.6(6)

BS EN 1993-1-5 NA.2.4

6.2.6(1)

Eq (6.17)

6.2.6(3)



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Therefore the shear resistance of the section is adequate.

1.8.3 Resistance to bending

Verify that:

$$\frac{M_{Ed}}{M_{c,Rd}} < 1.0$$

6.2.5(1)
Eq (6.12)

At the point of maximum bending moment (at B), verify whether the shear force will reduce the bending resistance of the cross section.

6.2.8(2)

$$\frac{V_{c,Rd}}{2} = \frac{756}{2} = 378 \text{ kN}$$

Shear force at maximum bending moment $V_{B,Ed} = 122 \text{ kN}$

$122 \text{ kN} < 378 \text{ kN}$

Therefore **no reduction** in bending resistance due to shear is required. (Low shear)

The design resistance for bending for Class 1 and 2 cross sections is:

6.2.5(2)

$$M_{c,Rd} = M_{pl,Rd} = \frac{W_{pl,y} \times f_y}{\gamma_{M0}} = \frac{1830 \times 10^3 \times 275}{1.0} \times 10^{-6} = 503 \text{ kN.m}$$

Eq (6.13)

$$\frac{M_{Ed}}{M_{c,Rd}} = \frac{382}{503} = 0.76 < 1.0$$

6.2.5(1)
Eq (6.12)

Therefore the bending moment resistance is adequate.

1.9 Buckling resistance of member in bending

If the lateral torsional buckling slenderness ($\bar{\lambda}_{LT}$) is less than or equal to $\bar{\lambda}_{LT,0}$ the effects of lateral torsional buckling may be neglected, and only cross-sectional verifications apply

6.3.2.2(4)

$\bar{\lambda}_{LT,0} = 0.4$ for rolled sections

NA.2.17

BS EN 1993-1-1 does not give a method for determining the elastic critical moment for lateral-torsional buckling (M_{cr}). Here a method presented in Access Steel document SN002 is used to determine a value for $\bar{\lambda}_{LT}$ without having to calculate M_{cr} .

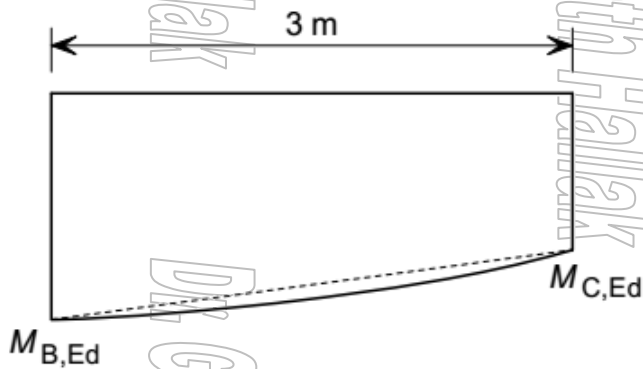
Access Steel
document
SN002



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Consider section B - C of the beam.



$$\bar{\lambda}_{LT} = \frac{1}{\sqrt{c_1}} UVD \bar{\lambda}_z \sqrt{\beta_w}$$

Calculate $1/\sqrt{c_1}$

End Moment Loading	ψ	$\frac{1}{\sqrt{c_1}}$	c_1	
	+1.00	1.00	1.00	
	+0.75	0.92	1.17	
	+0.50	0.86	1.36	
	+0.25	0.80	1.56	
	0.00	0.75	1.77	
	-0.25	0.71	2.00	
	-0.50	0.67	2.24	
	-0.75	0.63	2.49	
	-1.00	0.60	2.76	
			$\frac{1}{1.33 - 0.33\Psi}$	

$1/\sqrt{c_1}$ is a factor that accounts for the shape of the bending moment diagram

$$\psi = \frac{M_{C,Ed}}{M_{B,Ed}} = \frac{313}{382} = 0.819$$

$$\frac{1}{\sqrt{c_1}} = \frac{1}{1.33 - 0.33\psi} = \frac{1}{1.33 - 0.33 \times 0.819} = 0.94$$

$$g = \sqrt{\left(1 - \frac{I_z}{I_y}\right)} = \sqrt{\left(1 - \frac{1870}{37100}\right)} = 0.97$$



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$$U = \sqrt{\frac{W_{pl,y} \cdot g}{A} \frac{I_z}{I_w}} = \sqrt{\frac{1830 \times 10^3 \times 0.97}{104 \times 10^2} \frac{1870 \times 10^4}{0.922 \times 10^{12}}} = 0.877$$

$\beta_w = 1$ class 1 & 2

$$\lambda_z = \frac{kL}{i_z} = \frac{1 \times 3000}{42.3} = 70.92$$

$$\bar{\lambda}_z = \frac{\lambda_z}{\lambda_1} = \frac{70.92}{93.9 \times 0.92} = 0.821$$

$$V = \frac{1}{\sqrt[4]{1 + \frac{1}{20} \left(\frac{\lambda_z}{h/t_f}\right)^2}} = \frac{1}{\sqrt[4]{1 + \frac{1}{20} \left(\frac{70.92}{460/16}\right)^2}} = 0.936$$

D=1 load applied at the bottom flange

$$\bar{\lambda}_{LT} = 0.94 \times 0.877 \times 0.936 \times 1 \times 0.821 \times \sqrt{1} = 0.633$$

$0.633 > 0.4 (\bar{\lambda}_{LT,0})$

Therefore the resistance to lateral-torsional buckling must be verified

6.3.2.2(4)

$$\frac{M_{Ed}}{M_{b,Rd}} \leq 1.0$$

6.3.2.1(1)

Eq (6.54)

The design buckling resistance moment ($M_{b,Rd}$) of a laterally unrestrained beam is determined from:

6.3.2.1(3)

Eq (6.55)

$$M_{b,Rd} = \chi_{LT} \frac{W_y f_y}{\gamma_{M1}}$$

$W_y = W_{pl,y}$ for Class 1 and 2 cross-sections

For UKB sections, the method given in 6.3.2.3 for determining χ_{LT} for rolled sections may be used. Therefore

$$\chi_{LT} = \frac{1}{\phi_{LT} + \sqrt{\phi_{LT}^2 - \beta \bar{\lambda}_{LT}^2}} \text{ but } \begin{cases} \chi_{LT} \leq 1.0 \\ \chi_{LT} \leq \frac{1}{\bar{\lambda}_{LT}^2} \end{cases}$$

6.3.2.3(1)

Eq (6.57)

$$\phi_{LT} = 0.5 \left[1 + \alpha_{LT} (\bar{\lambda}_{LT} - \bar{\lambda}_{LT,0}) + \beta \bar{\lambda}_{LT}^2 \right]$$

From the UK National Annex, $\bar{\lambda}_{LT,0} = 0.4$ and $\beta = 0.75$

NA.2.17



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$$\frac{h}{b} = \frac{460.0}{191.3} = 2.4$$

$2 < 2.40 < 3.1$, therefore use buckling curve 'c'

For buckling curve 'c', $\alpha_{LT} = 0.49$

$$\phi_{LT} = 0.5 \left[1 + 0.49(0.633 - 0.4) + 0.75 \times 0.633^2 \right] = .707$$

$$\chi_{LT} = \frac{1}{0.707 + \sqrt{0.707^2 - 0.75 \times 0.633^2}} = 0.867$$

$$\frac{1}{\lambda_{LT}^2} = \frac{1}{0.633^2} = 2.5$$

$$0.867 < 1.0 < 2.5$$

Therefore,

$$\chi_{LT} = 0.867$$

To account for the shape of the bending moment distribution, χ_{LT} may be modified by the use of a factor 'f'

$$\chi_{LT,mod} = \frac{\chi_{LT}}{f} \text{ but } \chi_{LT,mod} \leq 1.0$$

$$f = 1 - 0.5(1 - K_c) \left[1 - 2.0(\lambda_{LT} - 0.8)^2 \right] \text{ but } f \leq 1.0$$

$$f = 1 - 0.5(1 - 0.94) \left[1 - 2.0(0.633 - 0.8)^2 \right] = 0.97$$

$$\chi_{LT,mod} = \frac{\chi_{LT}}{f} = \frac{0.867}{0.97} = 0.89$$

$$M_{b,Rd} = \chi_{LT,mod} \frac{W_y f_y}{\gamma_{M1}} = 0.89 \times 1830 \times 10^3 \times 275 \times 10^{-6} / 1.0 = 448 \text{ kN.m}$$

$$\frac{M_{A,Ed}}{M_{b,Rd}} = \frac{382}{448} = 0.85 \leq 1.0$$

Therefore the design buckling resistance moment of the member is adequate.

NA.2.17
NA.2.16 &
Table 6.5

6.3.2.3(2)

1.10 Vertical deflection at serviceability limit state

The vertical deflections should be verified.



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1.11 Resistance of the web to transverse forces

There is no need to verify the resistance of the web to transverse forces in this example, because the secondary beams are connected into the webs of the primary beams and flexible end plates are used to connect the beams to the columns.